RESERVE DEC - 6 2004

# GEORGIA INSTITUTE OF TECHNOLOGY

The George W. Woodruff
School of Mechanical Engineering

Ph.D. Qualifiers Exam - Fall Semester 2004

# SYSTEM DYNAMICS & CONTROL

**EXAM AREA** 

Assigned Number (DO NOT SIGN YOUR NAME)

\* Please sign your **name** on the back of this page —

# George W. Woodruff School of Mechanical Engineering Fall 2004 System Dynamics and Controls Doctoral Qualifying Examination

#### Instructions

There are 4 questions attached, please solve 3 of the four questions as completely as possible. State all assumptions, and make sure that you clearly indicate the thought processes that you employed to arrive at your answer. Answer only 3 questions. If you answer 4 questions, only the first 3 will be graded.

## TRANSFER FUNCTION ANALYSIS

The dynamic system of interest with input u(t) and response y(t) has the transfer function

$$\frac{Y(s)}{U(s)} = \frac{s+2}{(s+1)^2 + a^2}$$

Define a system to have overshoot to a step input at t = 0 if y(t), its response, reaches a maximum for a value of t where  $0 < t < \infty$ . For the system of interest we are concerned with the response of the system for values of a where  $-\infty < a^2 < \infty$ .

- a) For what values of a is the response oscillatory?
- b) For what values of a is the response unstable?
- c) For what values of a does the response have overshoot?

Justify your answers analytically. For your convenience a brief table of Laplace transforms is attached.

| Laplace   | transform           | naire |
|-----------|---------------------|-------|
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|-----|---|---|
|     | f(t)  | F(s)  |
| 1   | Unit impulse $\delta(t)$  | 1   |
| 2   | Unit step 1(t)  | $\frac{1}{y}$   |
| 3   | (   | $\frac{1}{s^2}$   |
| 4   | $\frac{t^{n-1}}{(n-1)!} \qquad (n=1,2,3,\ldots)$  | 1 57  |
| 5   | e <sup>-at</sup>  | $\frac{i}{s+a}$   |
| 6   | te -a!  | $\frac{1}{(s+a)^2}$                                       |
| 7   | $\frac{1}{(n-1)!}t^{n-1}e^{-\alpha t} \qquad (n=1,2,3,\ldots)$  | $\frac{1}{(s-a)^n}$                                       |
| 8   | $\frac{1}{b-a}\left(e^{-at}-e^{-bt}\right)$   | $\frac{1}{(s+a)(s+b)}$                                    |
| 9   | $\frac{1}{ab}\left[1+\frac{1}{a-b}\left(be^{-at}-ae^{-at}\right)\right]$  | $\frac{1}{s(s+a)(s+b)}$                                   |
| 10  | sin <i>at</i>   | $\frac{\omega}{s^2 + \omega^2}$                           |
| ! 1 | cos wr  | 32 + 402  |
| 12  | e <sup>−ar</sup> sin ωr   | $\frac{\omega}{(y+a)^2+\omega^2}$                         |
| 13  | e si cos wr   | $\frac{s+a}{(s+a)^2+\omega^2}$                            |
| 1.4 | $\frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta \omega_n t} \sin \omega_n \sqrt{1-\zeta^2} t$                        | $\frac{\omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2}$ |
| !5  | $-\frac{1}{\sqrt{1-\zeta^2}}e^{-\zeta\omega_n t}\sin\left(\omega_n\sqrt{1-\zeta^2}t+\phi\right)$                  |   |
|     | $\phi = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta}$   | $\frac{s}{s^2 + 2\zeta \omega_n s + \omega_n^2}$          |
| 16  | $1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta \omega_n t} \sin \left( \omega_n \sqrt{1 - \zeta^2} t + \phi \right)$ | $\omega_{\overline{z}}$                                   |
|     | $\phi = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta}$   | $\frac{\omega_n^2}{s(s^2+2\zeta\omega_n s+\omega_n^2)}$   |

## **ROOT LOCUS**

Consider the system shown in Figure 1, where the characteristics of the plant  $KG_p(s)$  is given by the root locus in Figure 2. Design a controller  $G_c(s)$  so that the system can operate

- (a) with a settling time that is one-quarter of the original system (where K=4.2), and
- (b) with a static velocity error constant at least 30 times larger than that of the original system (in order to ensure that the steady-state error to a ramp input is within an acceptable limit).

For the compensated system that you design, sketch the modified root locus and the response to a unit step and list the following results: the dominant complex poles, settling time, static velocity error constant, and maximum overshoot.

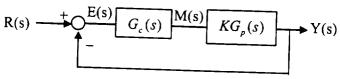


Figure 1

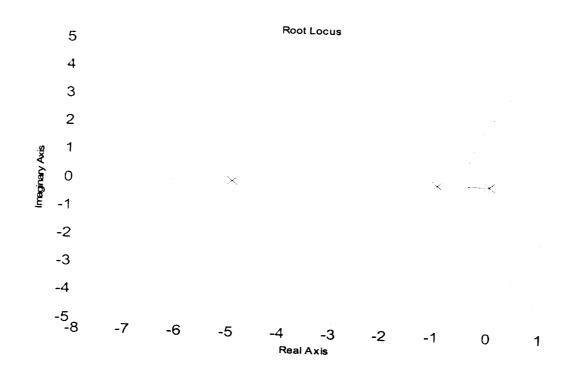


Figure 2

# FREQUENCY RESPONSE

Please use the following transfer function for this problem

$$\frac{Y(s)}{R(s)} = \frac{s+10}{s(s+0.1)^2(s+1000)}$$

#### Part a

Sketch the Bode plot for the transfer function. Please show gain in dB.

#### Part b

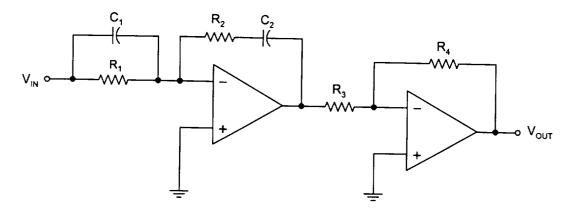
Determine the gain and phase margin for this system. Show them on your Bode Plot.

## Part c

Determine, to a reasonable approximation, y(t) when r(t) is given by

$$r(t) = 10\sin(0.1t) + 20\sin(t+0.1) + \sin(200t) + 0.1\sin(6000t)$$

## **MODELING**



- a. Determine the transfer function between the input voltage,  $V_{\text{in}}$ , and the output voltage,  $V_{\text{out}}$ .
- b. Can the analog electronic system shown in the above figure physically generate the transfer function that you derived for part a? This is a yes or no answer.
- c. Justify your answer to part b.