

# Dynamics Systems & Control Ph.D. Qualifying Exam Spring 2016

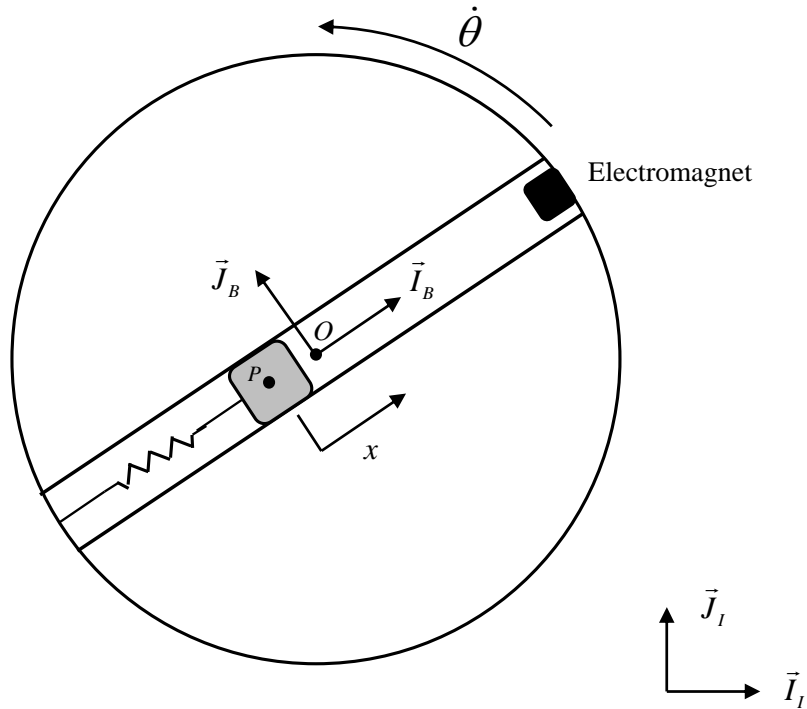
## **Instructions:**

Please work 3 of the 4 problems on this exam. It is important that you clearly mark which three problems you wish to have graded. For the 3 problems that you select, show all your work in order to receive proper credit. You are allowed to use a calculator.

Be sure to budget your time; concentrate on setting up the problem solution first and leave algebra until the end. When necessary, you may leave your answers in terms of unevaluated numerical expressions. Good Luck!

### Problem 1

A turntable of radius  $R$  rotates in the  $\vec{I}_I - \vec{J}_I$  plane as shown with a constant angular speed  $\dot{\theta}$ . A slot is cut in the turntable, and a slider of mass  $m$  slides within the slot. The variable  $x$  measures the position of the slider in the slot, where  $x = 0$  denotes that point  $P$  is located at the center of the turntable  $O$ . The slider is connected to a spring of unstretched length  $R$  (same as the radius of the turntable). Furthermore, the slider is subject to a frictional force exerted by the walls of the slot. Assume this frictional force can be modeled as viscous friction with a damping coefficient  $d$  (i.e., the frictional force along  $\vec{I}_B$  is given by  $-d\dot{x}$ ). The slider is magnetic, and is controlled via an electromagnet at one end of the slot which exerts a magnetic force on the slider  $u$  in the  $\vec{I}_B$  direction. Note that frame  $B$  rotates with the turntable and is centered at  $O$ , while frame  $I$  is an inertial reference frame. You may consider the slider to be a point mass for this problem.



- Find the transfer function  $G(s) = X(s)/U(s)$  from the control input to the position of the slider.
- Suppose you are designing this system and considering two possible sets of design parameters:

Design A:

$$\begin{aligned} \dot{\theta} &= 20 \text{ rad/sec} \\ m &= 2 \text{ kg} \\ k &= 1000 \text{ N/m} \\ d &= 5 \text{ N/(m/s)} \\ R &= 1 \text{ m} \end{aligned}$$

Design B:

$$\begin{aligned} \dot{\theta} &= 10 \text{ rad/sec} \\ m &= 2 \text{ kg} \\ k &= 200 \text{ N/m} \\ d &= 4 \text{ N/(m/s)} \\ R &= 1 \text{ m} \end{aligned}$$

Discuss qualitatively the response characteristics of the system if  $u$  is a unit step input when using Design A and Design B. What is the primary difference in the response characteristics between these two designs?

- Perform the same analysis as in b), except now let  $u$  be a unit impulse. What is the primary difference in the response characteristics between these two designs?

## Problem 2

Consider a unity-feedback system whose open-loop (feedforward) part is a proportional controller (P controller) followed by a third-order system

$$G(s) = \frac{s^2 + s + 1}{s^3 + s^2 + k_1 s + k_2}.$$

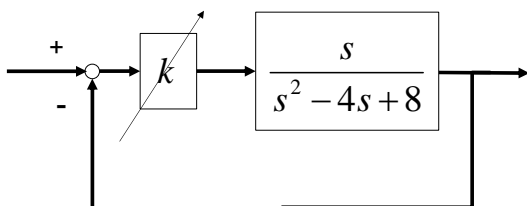
Here  $k_1$  and  $k_2$  are constant system parameters. Suppose  $k_1$  and  $k_2$  are unknown with

$$1 \leq k_1 \leq 2, \quad 3 \leq k_2 \leq 4.$$

- (a) Determine the range of the choice of the proportional gain  $K$  which guarantees the stability of the closed-loop system.
- (b) Choose  $K$  such that the magnitude of the steady-state error  $e_\infty$  of the system, where the reference input is a unit-step function, is guaranteed to be less than 0.01.

### Problem 3

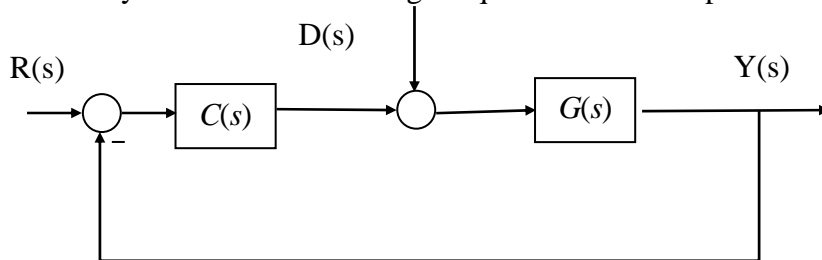
Consider the unity feedback of a dynamical system  $G(s) = \frac{s}{s^2 - 4s + 8}$  with a loop gain  $k$  ( $>0$ ) as shown below.



- Sketch the root-locus plot of the system. Determine the angles of departure from complex poles, intersection of asymptotes, break-in/away points, if they exist.
- (b-1) Determine  $k$  such that the closed-loop system is *undamped*, and (b-2) find the frequency of the undamped response of the closed-loop system.
- (c-1) Determine  $k$  such that the closed-loop system is *critically-damped*, and (c-2) sketch the unit-step response of the closed-loop system. (NOTE: a general sketch is acceptable and detailed mathematical derivations are not required. However, you should provide sufficient discussion to characterize the step response.)

### Problem 4

The Transfer function of the Segway transportation device shown below is given by  $\Theta(s)/U(s)=G(s)=s/(s^2-a^2)$  for some  $a>0$  where  $u$  is the linear velocity input and  $\theta$  is the angular displacement output You are asked to design a feedback controller  $C(s)$  (see the block diagram below) for this system to meet the design requirements to be specified.



- Is the open-loop system  $G(s)$  stable or not? Why?
- Design a controller of your choice (e.g., P,PD,PI,PID,Lead,Lag,...) such that the closed loop system has a settling time of at most  $4/a$  seconds, a Phase Margin (PM) of at least 60 degrees, and no steady state error for  $r=0$ .
- Find the steady state amplitude of  $y(t)$  in response to the sinusoidal input disturbance  $d(t)=d\cos\omega t$  for any  $\omega,d\geq 0$  and  $r=0$  using the controller you found in (b).

